# Kinematic and Dynamic Models

ME/CS 132b Advanced Robotics: Navigation and Perception 3/29/2011



#### Lecture Outline

Nomenclature

Kinematic Constraints and Models

Dynamic Constraints and Models

**Applications** 

Current and Active Research



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# Nomenclature

State Space 
$$\, {f X} \,$$
 Configuration Space  $\, {\cal C} \,$ 

Configuration q

(state-space)

Action u

Action Space 
$$\mathbf{U}(\mathbf{x})$$
  $\mathbf{U}(\mathbf{q})$ 

(state-space) (configuration-space)

$$\begin{array}{ll} \text{State/Configuration} & \dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}, \mathbf{u}) & \dot{\mathbf{q}} = \mathbf{f}(\mathbf{q}, \mathbf{u}) \\ \text{Transition Equation} & \dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}, \mathbf{u}) & \dot{\mathbf{q}} = \mathbf{f}(\mathbf{q}, \mathbf{u}) \end{array}$$



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## **Actions**

 ${f U}$  is the set of all possible actions over all states

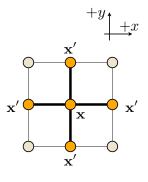
$$\mathbf{U} = \bigcup_{\mathbf{x} \in \mathbf{X}} \mathbf{U}(\mathbf{x})$$

Example: Moving on a 2D grid

$$\mathbf{x} = \begin{bmatrix} x & y \end{bmatrix}^T$$

$$x' = f(x, u) = x + u$$

$$\mathbf{U} = \{(0,1), (0,-1), (1,0), (-1,0)\}$$





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# Actions (cont.)

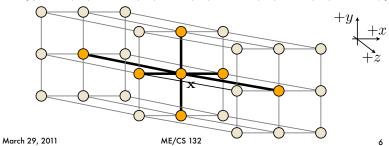
Example: Moving on a 3D grid  $\mathbf{x} = \left[ \begin{array}{ccc} x & y & z \end{array} \right]^T$ 

$$\mathbf{U} = \bigcup_{\mathbf{x}} \mathbf{U}(\mathbf{x})$$

$$\mathbf{x} = \begin{bmatrix} x & y & z \end{bmatrix}^T$$

 $\mathbf{x}' = \mathbf{f}(\mathbf{x}, \mathbf{u}) = \mathbf{x} + \mathbf{u}$  (same as before, just a higher dimensional state space)

$$\mathbf{U} = \{(0,0,1), (0,0,-1), (0,1,0), (0,-1,0), (1,0,0), (-1,0,0)\}$$



# Actions (cont.)

In continuous spaces the next state is determined by a integrating the state transition equation

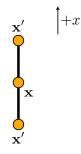
$$\mathbf{U} = \bigcup_{\mathbf{x} \in \mathbf{X}} \mathbf{U}(\mathbf{x})$$

Example: One-dimensional particle

$$\mathbf{x} = \left[x\right]^T$$

$$\mathbf{\dot{x}} = v_x$$

$$\mathbf{U} = \{v_x\} = [-1, 1]$$





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#### **Kinematic Constraints**

$$\mathbf{g}(\mathbf{q}, \mathbf{\dot{q}}) \bowtie 0$$

$$\dot{\mathbf{q}} = \mathbf{f}(\mathbf{q}, \mathbf{u})$$

$$g_1(\mathbf{q})\dot{q_1} + g_2(\mathbf{q})\dot{q_2} + \ldots + g_n(\mathbf{q})\dot{q_n} = 0$$



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## Kinematic Constraints (cont.)

Pfaffian 
$$g_1(\mathbf{q})\dot{q_1} + g_2(\mathbf{q})\dot{q_2} + \ldots + g_n(\mathbf{q})\dot{q_n} = 0$$

# Configuration transition equation

$$\dot{q_1} = u_1 \qquad q_{n-k+1} = f_{n-k+1}(\mathbf{q}, \mathbf{u})$$
 $\dot{q_2} = u_2 \qquad q_{n-k+2} = f_{n-k+2}(\mathbf{q}, \mathbf{u})$ 
 $\dot{q_{n-k}} = u_{n-k} \qquad \dot{q_n} = f_n(\mathbf{q}, \mathbf{u})$ 

(these are determined by solving for the remaining variables)



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## Kinematic Constraints (cont.)

#### A quick example...

$$\mathcal{C}=\mathbb{R}^3 \text{ (three dimensional state space, n=3)}$$
 
$$7\dot{q}_1-3\dot{q}_2+\dot{q}_3=0 \text{ (one Pfaffian constraint, k=1)}$$
 
$$(\text{two control inputs (n-k=2)}$$
 
$$\dot{q}_1=u_1 \qquad \qquad \dot{q}_1=u_1$$
 
$$\dot{q}_2=u_2 \qquad \qquad \dot{q}_2=u_2$$
 
$$(\text{substituting for configuration rates}) \qquad \dot{q}_3=-7u_1+3u_2$$
 
$$7u_1-3u_2+\dot{q}_3=0 \qquad \text{(configuration transition equation)}$$



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## Simple Car Model

Rigid body that moves in a twodimensional configuration space

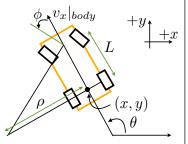
$$\mathcal{C}=\mathbb{R}^2\times\mathbb{S}^1$$

$$q = \begin{bmatrix} x & y & \theta \end{bmatrix}^T$$

$$\dot{x} = f_1(x, y, \theta, v_x|_{body}, \phi)$$

$$\dot{y} = f_2(x, y, \theta, v_x|_{body}, \phi)$$

$$\dot{\theta} = f_3(x, y, \theta, v_x|_{body}, \phi)$$





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# Simple Car Model

Rigid body that moves in a twodimensional configuration space

$$\mathcal{C} = \mathbb{R}^2 imes \mathbb{S}^1$$

$$q = \begin{bmatrix} x & y & \theta \end{bmatrix}^T$$

$$\frac{dy}{dx} = tan(\theta) = \frac{sin(\theta)}{cos(\theta)} = \frac{\dot{y}}{\dot{x}} \text{ as } t \leftarrow 0 \quad \phi v_x|_{body}$$

$$-\dot{x}sin(\theta) + \dot{y}cos(\theta) = 0$$

(rewritten as a Pfaffian constraint)

$$\dot{x} = cos(\theta) = v_x|_{body}cos(\theta)$$

$$\dot{y} = sin(\theta) = v_x|_{body}sin(\theta)$$



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# Simple Car Model

#### How are these constraints found?

$$\begin{aligned} -\dot{x}sin(\theta) + \dot{y}cos(\theta) &= 0 \\ \text{(n=2, k=1)} & \downarrow \\ \dot{x} &= u_1 \\ \dot{y} &= u_1tan(\theta) \\ & \downarrow \text{(substituting for u_1)} \\ \dot{x} &= cos(\theta) \\ \dot{y} &= cos(\theta) \frac{sin(\theta)}{cos(\theta)} = sin(\theta) \end{aligned}$$

 $\mathcal{C} = \mathbb{R}^2 \times \mathbb{S}^1$  $\phi v_x|_{body}$ 

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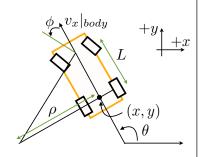
# Simple Car Model

Rigid body that moves in a twodimensional configuration space

$$\mathcal{C} = \mathbb{R}^2 \times \mathbb{S}^1$$

$$q = \begin{bmatrix} x & y & \theta \end{bmatrix}^T$$

(distance traveled) 
$$d\dot{\omega}=\rho d\theta$$
 
$$d\theta=\frac{tan(\phi)}{L}d\omega$$
 (divide by dt)  $\dot{\theta}=\frac{v_x|_{body}}{L}tan(\phi)$ 





## Simple Car Model

#### With control inputs of the form

$$u = (u_{v_x|_{body}}, u_{\phi})$$

# the configuration transition equation is

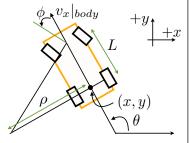
$$\dot{x} = u_{v_x|_{body}} cos(\theta)$$

$$\dot{y} = u_{v_x|_{body}} sin(\theta)$$

$$\dot{\theta} = \frac{u_{v_x|_{body}}}{L} tan(u_{\phi})$$



$$q = \begin{bmatrix} x & y & \theta \end{bmatrix}^T$$





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## Variations of the Simple Car Model

#### Simple Car

$$U = [-1,1] imes (-\phi_{max},\phi_{max})$$
 (cannot turn in place)

#### Tricycle

$$U = [-1,1] imes (-rac{\pi}{2},rac{\pi}{2})$$
 (can turn in place)

#### Reed-Shepp Car

$$U = \{-1,0,1\} \times (-\phi_{max},\phi_{max}) \text{ (can drive forwards and backwards)}$$

#### **Dubins Car**

$$U = \{0,1\} imes (-\phi_{max},\phi_{max})$$
 (cannot drive backwards)



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### Differential Drive Model

With inputs of the form

$$\mathcal{C} = \mathbb{R}^2 \times \mathbb{S}^1$$

$$u = (u_{\omega_r}, u_{\omega_l})$$

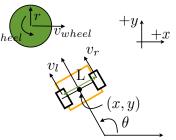
$$q = \begin{bmatrix} x & y & \theta \end{bmatrix}^T$$

The configuration transition equation is

$$\dot{x} = \frac{r}{2}(u_{\omega_l} + u_{\omega_r})cos(\theta)$$

$$\dot{y} = \frac{r}{2}(u_{\omega_l} + u_{\omega_r})sin(\theta)$$

$$\dot{\theta} = \frac{r}{L}(u_{\omega_r} - u_{\omega_l})$$





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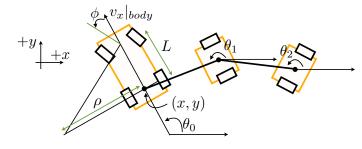
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## N-Trailers

A variation of the simple car model involves one that is pulling N passively controlled  $\ q = \left[ \begin{array}{ccccc} x & y & \theta_0 & \theta_1 & \theta_2 \end{array} \right]^T$ trailers

$$\mathcal{C} = \mathbb{R}^2 \times \mathbb{S}^1 \times \mathbb{S}^1 \times \mathbb{S}^1$$

$$q = \begin{bmatrix} x & y & \theta_0 & \theta_1 & \theta_2 \end{bmatrix}^T$$





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## **Dynamic Constraints**

Implicit

$$\mathbf{g_i}(\ddot{\mathbf{q}}, \dot{\mathbf{q}}, \mathbf{q}) = 0$$

similar to the kinematic constraints

**Parametric** 

$$\mathbf{\ddot{q}} = \mathbf{f}(\mathbf{\dot{q}}, \mathbf{q}, \mathbf{u})$$



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## **Phase Space**

Increase the dimension to remove higher-order derivatives

define a phase vector

$$\mathbf{x} = (x_1, x_2)$$

set the values of the phase vector

a single constraint

$$4\ddot{y} - 5\dot{y} + y = 0$$

 $x_1 =$ 

creates new constraints

$$x_2 = \dot{y} \qquad \dot{x_2} = \ddot{y}$$
$$\dot{x_1} = x_2$$

system is converted into two first-order differential equations

$$\dot{z}_1 \stackrel{\searrow}{=} x$$

$$\begin{array}{c|c} x_2 \\ 5x_2 - x_1 \end{array}$$

$$\dot{c}_2 = \frac{\partial x_2}{\partial x_2}$$



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## Improved Car Model

Use a double integrator to ensure a continuously varying steering angle

$$\mathbf{x} = \begin{bmatrix} x & y & \theta & \phi \end{bmatrix}^T$$

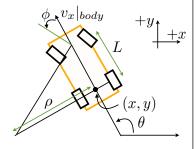
$$\mathbf{u} = (u_{v_x|_{body}}, u_{\omega})$$

$$\dot{x} = v_x|_{body}cos(\theta)$$

$$\dot{y} = v_x|_{body}sin(\theta)$$

$$\dot{\theta} = \frac{v_x|_{body}}{L}tan(\phi)$$

$$\dot{\phi} = u_{\omega}$$





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# Improved Differential Drive Model

This can similarly be applied for the differential drive model

$$\mathbf{x} = \begin{bmatrix} x & y & \theta & \omega_l & \omega_r \end{bmatrix}^T$$

$$\mathbf{u} = (u_{\alpha_l}, u_{\alpha_r})$$

$$\dot{x} = \frac{r}{2}(\omega_l + \omega_r)\cos(\theta)$$

$$\dot{y} = \frac{r}{2}(\omega_l + \omega_r)sin(\theta)$$

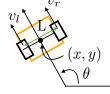
$$\dot{\theta} = \frac{r}{L}(\omega_r - \omega_l)$$

$$\mathbf{x} = \begin{bmatrix} x & y & \theta & \omega_l & \omega_r \end{bmatrix}^T$$

$$v_{wheel}$$
  $v_{r}$   $v_{r}$   $v_{r}$ 

$$\dot{\omega}_l = u_\alpha$$

$$\dot{\omega_r} = u_{\alpha_r}$$





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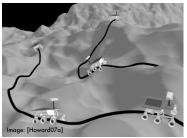
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# **Predictive Model Applications**

Predictive Terrain Compensation [Howard07a]

Exploited a motion model that simulated the effects of terrain geometry to plan paths that considered rough terrain and wheel slip models







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# Predictive Model Applications (cont.)

Autonomous automobile navigation [Ferguson08a]

Modeled curvature, curvature rate, safety controller constraints







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### Current and Active Research

Online Slip Identification [Rogers-Marcovitz10a]

Extended Kalman filter to calibrate a parameterized predictive motion model in real-time

Planar slip disturbances modeled as second-order functions of velocity and curvature

Demonstrated on multiple platforms

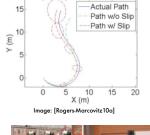




Image: [Rogers-Marcovitz10



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#### Next

Lecture 2: Motion simulation (3/31)

**Newton-Euler Mechanics** 

Motion of Particles

Motion Simulation (Linear, Euler, Runge-Kutta)

Homework #1 Assigned



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## **Document/Image References**

[Ferguson08a] David Ferguson, Thomas Howard, and Maxim Likhachev, "Motion Planning in Urban Environments: Part I," Proceedings of the IEEE/RSJ 2008 International Conference on Intelligent Robots and Systems, September, 2008.

[Howard07a] Thomas Howard and Alonzo Kelly, "Optimal Rough Terrain Trajectory Generation for Wheeled Mobile Robots," International Journal of Robotics Research, Vol. 26, No. 2, March, 2007, pp. 141-166.

[Howard 09a] T.M. Howard, "Adaptive Model-Predictive Motion Planning for Navigation in Complex Environments". Ph.D. Thesis, Carnegie Mellon University, August 2009 [LaValle 06a] S. LaValle, "Planning Algorithms". Cambridge: Cambridge University Press. ISBN 0521862051.

[Rogers-Marcovitz10a] Forrest Rogers-Marcovitz and Alonzo Kelly, "On-line Mobile Robot Model Identification using Integrated Perturbative Dynamics," 12th International Symposium on Experimental Robotics, December, 2010.



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