



CDS 110b: Lecture 9-1 Robust Stability



Richard M. Murray
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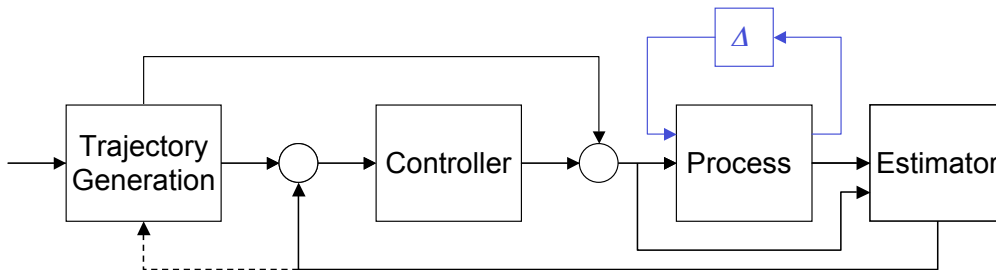
Goals:

- Describe methods for representing unmodeled dynamics
- Derive conditions for robust stability

Reading:

- AM06, Section 9.5, 12.1, 12.2 (norms + robust stability)
- Advanced: Doyle, Francis and Tannenbaum, Sections 4.1-4.3

Game Plan: Robust Performance



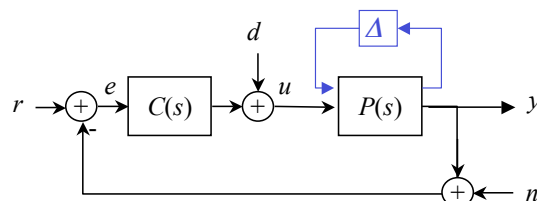
Robust Control

- Stability: bounded inputs \rightarrow bounded outputs
- Performance: keep errors small

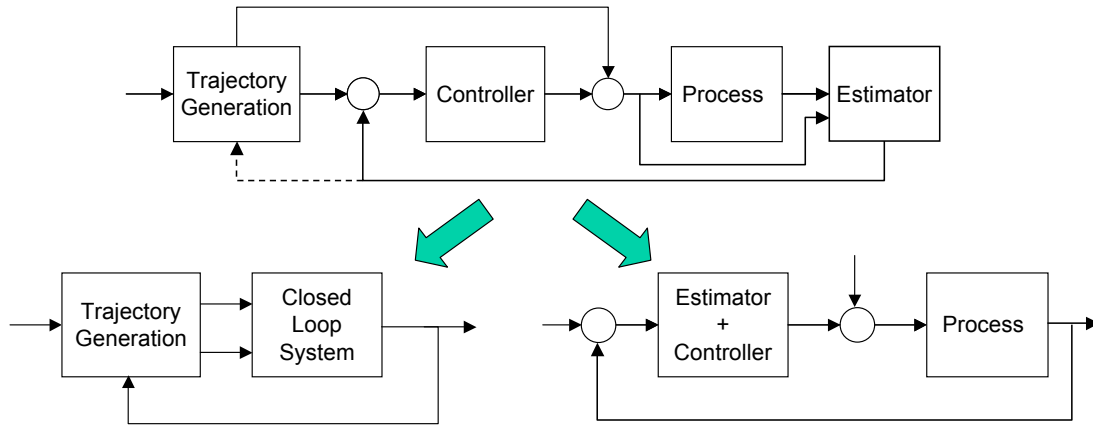
Robustness: do these things for all $\|\Delta\|_\infty < 1$

Simplifying case: focus on basic control loop (inner or outer loop)

- Stability = "internal stability"
- Performance = bounds on S & T
- Robustness = small gain theorem



Inner/Outer Loop Analysis



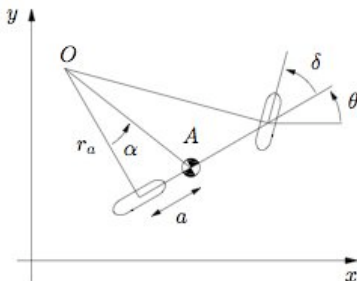
Outer loop: assume tracking is fast

- Replace tracking controller with closed loop dynamics
- Assumes tracking is much faster than trajectory generation
- Remaining uncertainty from system objective (ref) + environmental change

Inner loop: assume reference is slow

- Treat reference as constant input
- Design controller to provide sufficient bandwidth, tracking, disturbance rej.
- Reduces to standard (CDS 110a) control design; bandwidth requirements based on outer loop

Motivating Example: Vehicle Steering (AM06)



$$\begin{aligned} \dot{\xi} &= v \cos(\alpha + \theta) & \frac{dw}{d\tau} &= \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} w + \begin{bmatrix} \alpha \\ 1 \end{bmatrix} u \\ \dot{\eta} &= v \sin(\alpha + \theta) & y &= \begin{bmatrix} 1 & 0 \end{bmatrix} w \\ \dot{\theta} &= \frac{v_0}{b} \tan \delta, \end{aligned}$$

- Control the lateral dynamics to follow trajectory
- Normalize dynamics to simply functional form (AM06)

Control design

$$u = -k_1 w_1 - k_2 w_2 + k_r r,$$

- Choose gains so that closed loop dynamics have characteristic polynomial

$$p(s) = s^2 + 2\zeta_c \omega_c s + \omega_c^2.$$

$$\omega_c = 1, \zeta_c = 0.707$$



$$k_1 = 100 \text{ and } k_2 = -35.86$$

Estimator design

$$\frac{d\hat{w}}{dt} = A\hat{w} + Bu + L(y - C\hat{w})$$

- Choose estimator gains to give closed loop observer with characteristic poly

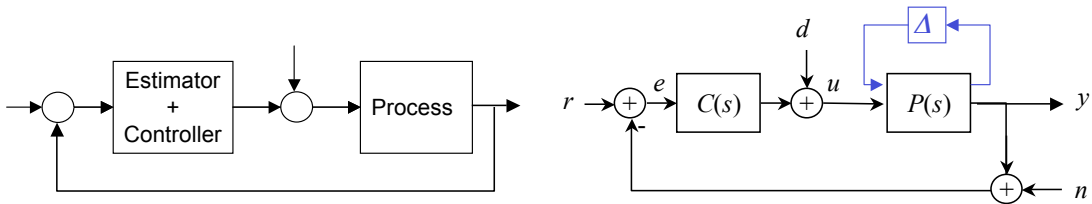
$$q(s) = s^2 + 2\zeta_o \omega_o s + \omega_o^2,$$

$$\omega_o = 20 \text{ and } \zeta_o = 2$$



$$l_1 = 28.28 \text{ and } l_2 = 400$$

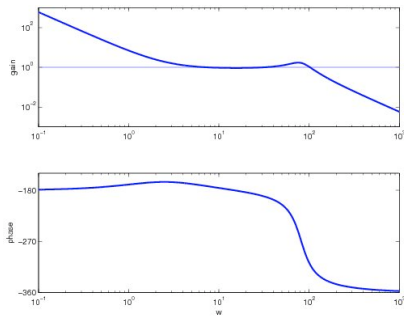
Closed Loop Steering Dynamics



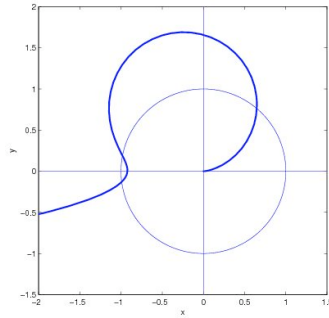
Process and controller dynamics

$$P(s) = \frac{0.5s + 1}{s^2}, \quad C(s) = \frac{-11516s + 40000}{s^2 + 42.4s + 6657.9}$$

Bode Plot



Nyquist plot



Closed loop not robust!

- No net encirclements, but phase margin is very small (around 7 deg)
- Small change in process dynamics will cause system to go unstable
- Not clear from design what went wrong - closed loop dynamics look reasonable
- Problem is not limited to eigenvalue placement (HW: show for LQR + KF)

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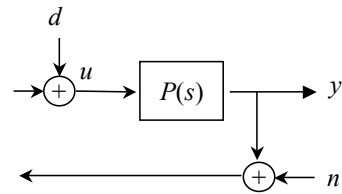
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Modeling Uncertainty

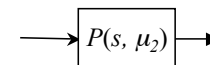
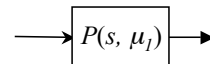
Noise and disturbances

- Model the amount of noise by its signal strength in different frequency bands
- Can model signal strength by peak amplitude, average energy, and other norms
- Typical example: Dryden gust models (filtered white noise)



Parametric uncertainty

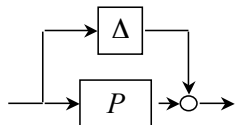
- Unknown parameters or parameters that vary between plants
- Typically specified as tolerances on the nominal parameters



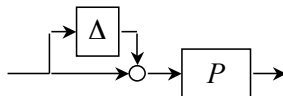
Unmodeled dynamics

- High frequency dynamics can be excited by control loops
- Use bounded operators to account for unmodeled dynamics:

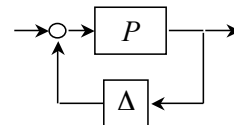
Additive uncertainty



Multiplicative uncertainty



Feedback uncertainty



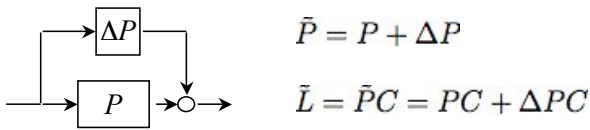
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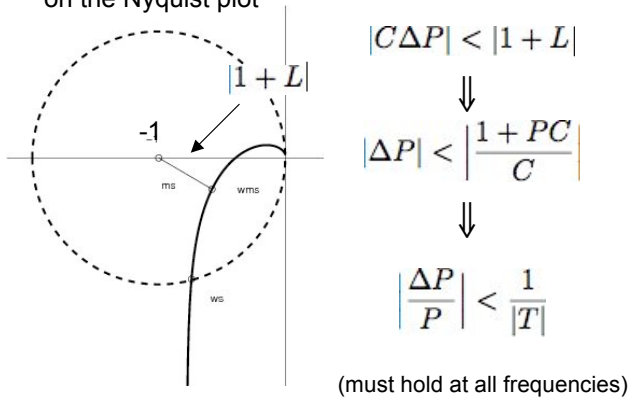
Robust Stability via Nyquist

Additive uncertainty case

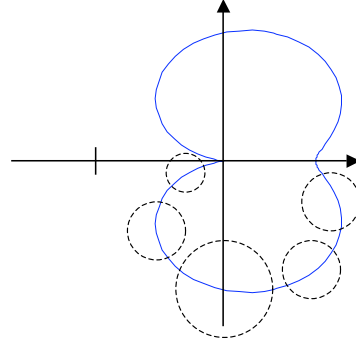


How much ΔP can we tolerate?

- Look at distance from the critical point on the Nyquist plot

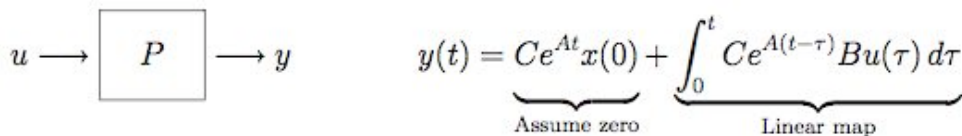


“Fuzzy” Nyquist plot



- Can handle any uncertainty within “tube” around nominal process (no new encirc’ts)
- Caution: requires perturbations to be *stable*
- Conservative condition: allows *any* variation w/in tube; reality may be more kind

System Norms



What is the “gain” from u to y ?

- Would like a single number, not something dependent on frequency
- Answer depends on what you choose as the norms for the input and outputs

2-norm of a signal

- Define like the two norm of a vector (but with integral instead of a sum):

$$\|u\|_2 = \left(\int_{-\infty}^{\infty} u^2(t) dt \right)^{1/2}$$

- Corresponds to the “energy” contained in a signal
- Caution: not defined for sinusoids (!)

Induced norm of a system

- Look at maximum norm of the output given all possible unit-norm inputs

$\ u\ _2 = \left(\int_{-\infty}^{\infty} u^2(t) dt \right)^{1/2}$	$\ u\ _{\infty} = \sup_t u(t) $
$\ y\ _2 = \sup_{\omega} H(j\omega) $	∞
$\ y\ _{\infty} = \frac{1}{2\pi} \left(\int_{-\infty}^{\infty} H(j\omega) ^2 d\omega \right)^{1/2}$	$\int_{-\infty}^{\infty} C e^{At} B dt$

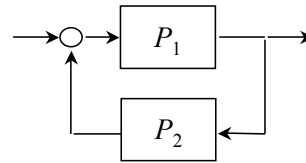
- Common choice for control purposes is the “induced 2-norm”

$$\|H\|_{\infty} = \sup_{\omega} |H(i\omega)|.$$

also called the “infinity norm” (of a transfer function)

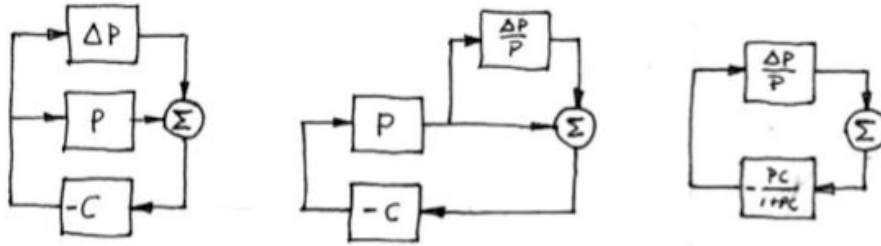
Small Gain Theorem

Theorem (Small gain theorem) Consider two stable, linear time invariant processes with transfer functions $P_1(s)$ and $P_2(s)$. The feedback interconnection of these two systems is stable if $\|P_1 P_2\|_\infty < 1$.



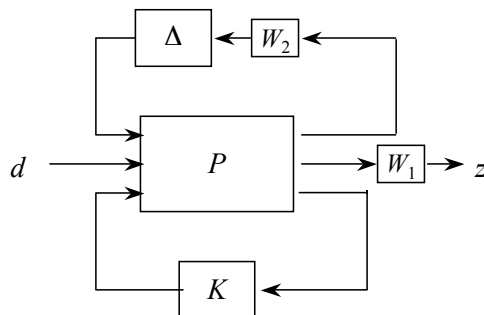
Application to robust stability

- Use block diagram algebra to convert system into form of small gain theorem



$$\left\| \frac{\Delta P}{P} \cdot T \right\|_\infty < 1 \implies \left| \frac{\Delta P}{P} \right| < \frac{1}{|T|} \quad \forall \omega$$

Preview: Robust Performance



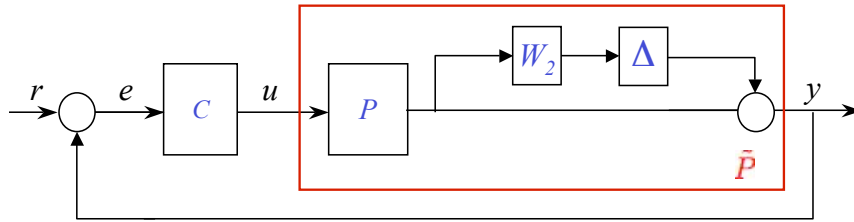
- d disturbance signal
- z output signal
- Δ uncertainty block
- W_2 performance weight
- W_1 uncertainty weight

Goal: guaranteed performance in presence of uncertainty

$$\|z\|_2 \leq \gamma \|d\|_2 \quad \text{for all } \|\Delta\| \leq 1$$

- Compare energy in disturbances to energy in outputs
- Use frequency weights to change performance/uncertainty descriptions
- “Can I get X level of performance even with Y level of uncertainty?”

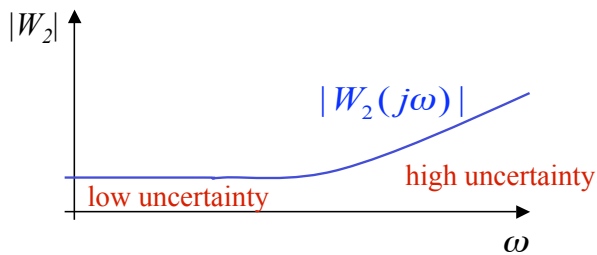
Robust Stability Using Frequency Domain Weighting



Multiplicative Uncertainty

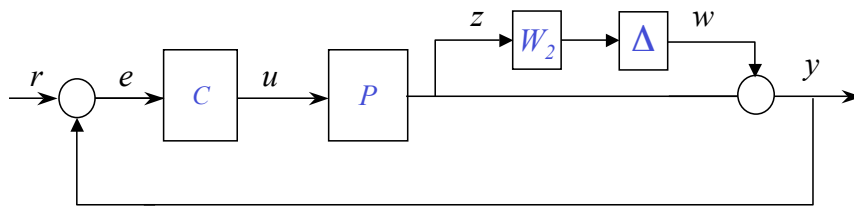
- Model plant as nominal with additional dynamics given by $W_2 \Delta$

$$\tilde{P} = P(1 + W_2 \Delta) \quad \begin{array}{l} W_2 = \text{frequency weight} \\ \Delta = \text{uncertainty; require } |\Delta(j\omega)| \leq 1 \end{array}$$



- Δ allows *any* dynamics to be inserted into the plant
- Can be used to model parameter uncertainty or unmodeled dynamics

Complementary Sensitivity and Robustness



Thm A controller C provides robust stability to multiplicative perturbations if and only if

$$|W_2(j\omega) T(j\omega)| < 1 \quad \text{for all } \omega.$$

where

Complementary
sensitivity
function

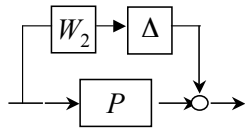
$$T := \frac{PC}{1 + PC} = H_{zw}$$

Intuition: H_{zw} represents the transfer function seen by the weighted uncertainty $W_2 \Delta$

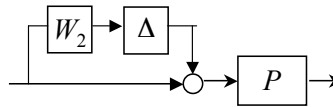
Note: this theorem guarantees stability for any *transfer function* $\Delta(s)$ satisfying $|\Delta(j\omega)| < 1 \Rightarrow$ allows unmodeled *dynamics* (as well as parametric uncertainty)

Models for Uncertainty

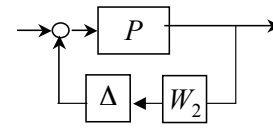
Additive uncertainty



Multiplicative uncertainty



Feedback uncertainty



Each model describes a class of process dynamics:

- Additive: $\tilde{P} = P + W_2\Delta$
 - Multiplicative: $\tilde{P} = P(1 + W_2\Delta)$
 - Feedback: $\tilde{P} = P/(1 + PW_2\Delta)$
- } Use $\|W_2\|$ to shape the unmodeled dynamics; $\|\Delta\|_\infty < 1$ in all cases

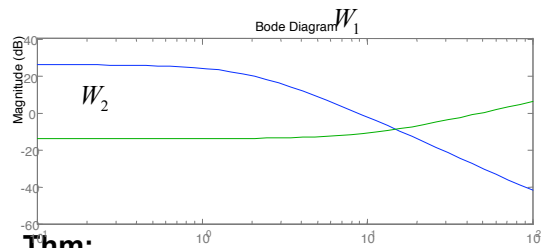
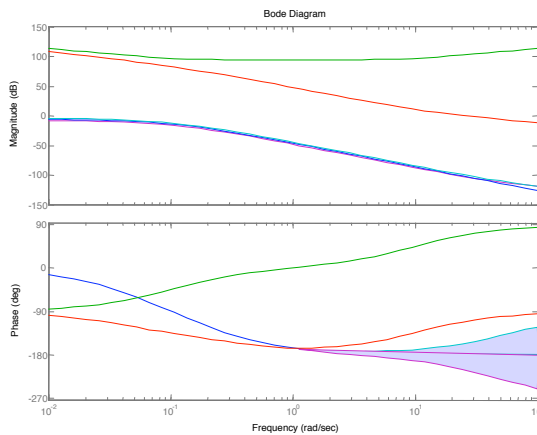
Robust stability conditions given by small gain theorem

- Compute transfer function around Δ block and require that this be < 1
- (If not, can choose Δ with $\|\Delta\|_\infty \leq 1$ to destabilize)

Example: Robust Cruise Control



$$P(s) = \frac{1/m}{s + b/m} \cdot \frac{r}{s + a} (1 + W_2\Delta)$$



Thm:

- Performance: $\left| W_1 \frac{1}{1+L} \right| < 1 \quad \forall \omega$
- Robust Stability: $\left| W_2 \frac{L}{1+L} \right| < 1 \quad \forall \omega$

