# Announcements:

1. Final available, due W 3119 by Spm 2. HW # 15 graded by Thursday pm, solh's posted

CDS 110b Final Review Notes

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3. Hw # 16 graded by Mon, , soli's posted this Friday .

4. Please fill out surveys &

## 1 General Comments

The exam will cover mostly material since the midterm, but may include or rely on concepts from the first half of the term. As usual, the best resources for review are

- Past Homeworks... #'s 12-16
- Lecture Notes...
- Text Reading and Examples...

# 2 DFT Chapter 2: Norms for Signals and Systems

In this chapter, the temporal norms of time-varying signals are presented as a way to measure their size. Also, the notion of a norm for a system's transfer function is introduced. Consider two signals  $e_1(t)$  and  $e_2(t)$ :



For  $e_1(t)$  the infinity-norm (peak) is 1,  $||e_1(t)||_{\infty} = 1$ , whereas since the signal does not "die out" the 2-norm is infinite,  $||e_1(t)||_2 = \infty$ . For  $e_2(t)$  the opposite is true.

Consider a system G with input d and output e,

### e = Gd

For performance we may want the output signal e to be small for any allowed input signals d, i.e. we need to specify the set that d belongs to and what we mean by "small". Possible input signals

<sup>\*</sup>These notes adapted from "CDS 212 Revision notes", parts 1 and 2 by Antonis Papachristodoulou. Discrete time system notes contributed by Zhipu Jin. DFT refers to Doyle, Francis and Tannenbaum. *Feedback Control Theory*. Macmillan. 1992.

sets might be specific (may be white noise process with zero mean, a sinusoid of fixed frequency etc) or they might be specified by norm bounds (bounded in energy (2-norm) in power (power "norm"), in magnitude ( $\infty$ -norm)). To measure the output signal one may consider the 1-norm, the 2-norm (energy), the  $\infty$ -norm or the power.

The question to ask is what is the appropriate system gain to test for performance. You should be familiar with induced norms and translating signal specifications into equivalent specifications on system norms.

**Example 2.1.** Let u(t) be a continuous signal whose derivative  $\dot{u}(t)$  is also continuous. Which of the following qualify as a norm for u?

•  $\sup_{t} |\dot{u}(t)|$   $u(t) = 1 \implies ||u|| = 0$  vio |axes (ii) ii)  $||u|| = 0 \implies u(t) \equiv 0$ •  $|u(0)| + \sup_{t} |\dot{u}(t)|$  iii)  $||u|| = 0 \implies u(t) = ||u|| + ||v|||$ (i)  $||u|| = 0 \implies u(t) \implies u(t)$ 

### 3 DFT Chapter 3: Basic Concepts

**Standard Block Diagrams** For a plant P, controller C, disturbance d, noise n, ...



**Theorem 3.1 (Nyquist Stability Criterion).** Let N be the number of poles of G(s) in the RHP. Then  $\frac{1}{1+\alpha G}$  is stable iff the number of anticlockwise encirclements of the  $-1/\alpha$  point equals N.

Internal Stability Consider the plant

$$P(s) = P_1(s)\frac{2-s}{2+s}$$

and a proposed controller

$$C(s) = C_1(s)\frac{2+s}{2-s}$$

where  $C_1$  is a suitable compensator for  $P_1$ . Now  $PC = P_1C_1$ , so if  $\frac{1}{1+P_1C_1}$  is stable,  $\frac{1}{1+PC}$  will be too. However we must ensure that the response of the control input to disturbances is also stable, and in this case we find that

$$\frac{C}{1+CP} = \frac{C_1}{1+C_1P_1}\frac{2+s}{2-s}$$

which is unstable. In general, a feedback system comprising a plant P and a controller C is internally stable if each of the following closed-loop transfer functions is stable:

$$\begin{pmatrix} x_1 \\ x_2 \\ x_3 \end{pmatrix} = \frac{1}{1+Pcf} \begin{pmatrix} 1 & -Pf & -f \\ c & 1 & -cf \\ Pc & P & 1 \end{pmatrix} \begin{pmatrix} r \\ d \\ n \end{pmatrix}$$

In fact, we only need to check if two of them are. Which two?

This condition is equivalent to any one of them being stable (i.e. 1 + PC having no zeros in the RHP) and there being no closed RHP zero/pole cancellations between P and C.

## 4 DFT Chapter 4: Uncertainty and Robustness

Recall that making the sensitivity function small is one of the most fundamental objectives of feedback design. Consider T(s) as a function of P. Then the closed loop transfer function (complementary sensitivity)

$$T = \frac{PC}{1 + PC}$$

has some perturbation due to infinitesimal changes in the plant which is

$$\lim_{\Delta P \to 0} \frac{\Delta T/T}{\Delta P/P} = \frac{dT}{dP} \frac{P}{T} = \frac{(1+pt)a - pcte)}{(1+pc)x} \frac{b}{pba} = \frac{1}{1+pc}$$

$$= S \quad (sensitiv: ty fraction)$$

This means that the relative error in the closed-loop transfer function is reduced compared to the relative error in the open loop transfer function if  $||S(s)||_{\infty} < 1$ , or when we introduce a weight,  $||W_1S||_{\infty} < 1$ . This requires that at every frequency the point  $L(j\omega)$  on the Nyquist plot lies outside the disk of center -1, radius  $W_1(j\omega)$ .



$ w_1 S(jw)  < 1$ $\Leftrightarrow  S(jw)  < \frac{1}{ w_1 }$	1m,1=151 (L(jav)
C) 1 ISCIWI > IWI	PP
(=> 11+L1 > 1w1/	J N

**Example 4.1.** We all know the notion of phase and gain margin. True or false?  $||S(s)||_{\infty}$  will become large if and only if our system has poor gain margin or poor phase margin.



We will use the Small Gain Theorem to find a condition for robust stability:

**Theorem 4.1.** Suppose G(s) is a stable fixed transfer function. Let  $\gamma$  be a positive constant. Then the feedback system

H	4	
-	6	H

is stable for all stable  $\Delta(s)$  satisfying  $|\Delta(j\omega)| \leq \gamma$  iff  $||G(j\omega)||_{\infty} < 1/\gamma$ .

Consider now a multiplicative uncertainty model, assuming the feedback system is stable. Write

$$\tilde{P} = (1 + W_2 \Delta) P$$

where  $W_1, P$  are fixed,  $\Delta$  is not, but stable and norm bounded:  $\|\Delta\|_{\infty} \leq 1$ .

Example 4.2. Multiplicative uncertainty.



$$x = -PC(x + w_{2}y)$$

$$\Rightarrow x(1+PC) = -w_{2}PCy$$

$$\Rightarrow \frac{x}{y} = -w_{2}T = -G$$

$$\Rightarrow G = w_{2}T$$

$$\Rightarrow Robust stadius TY$$

11 WZTH C1

Recall the following constraints:

PERFORMANCE: 11 W, S 110 < 1

RS: 1/W2THD <1

NOM

RP 🔿

 $\mathbf{5}$ 

• Algebraic Constraints:

1.  $\mathbf{S} + \mathbf{T} = \mathbf{1}$ : At any frequency  $\omega$ ,  $|S(j\omega)|$  and  $|T(j\omega)|$  cannot be both less than 0.5.

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111W, SI + IW\_TI 11 = <1 (PRODE IN DET SEC 4.6)

2. The weighting functions  $W_1(j\omega)$  and  $W_2(j\omega)$  satisfy

DFT Chapter 6: Design Constraints

 $m: n \in [W_1(jw)], [W_2(jw)] = 2$ 

Typ: Win

Recall that  $W_1$  is monotonically decreasing for good tracking of low-frequency signals, and  $W_2$  is monotonically increasing, as uncertainty increases at high frequencies.

3. If p is a pole and z is a zero of L in the RHP, then

$$S(p) = 0$$
  $S(z) = 1$   
 $T(p) = 1$   $T(z) = 0$ 

nonminimum phase

- Analytic Constraints:
  - 1. If L has a zero z in RHP then
- $\|W_1 S\|_{\infty} \ge |W_1(z)|$
- 2. If L has a pole p in RHP then

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$$\left\|W_{2}T\right\|_{\infty} \geq \left\|W_{2}\left(p\right)\right\|$$

3. If L has a pole p in RHP and a zero z in RHP then

Cansider case where O  $S = Sap Smp = no RAP zeros <math>Sap = \frac{S-p}{S+p}$ th only one  $ISap|=1 \forall w$ Tap =  $\frac{S-2}{S+2}$ yole in RMP T = Tap Tmp

$$||w_{i} S||_{\infty} = ||w_{i} S_{mp}||_{\infty} = ||w_{i} S_{ap}^{-i}||_{\infty} \geq ||w_{i}(z) S_{ap}(z)| = ||w_{i}(z)|| \frac{z+p}{z-p}|$$

$$||w_{2} T||_{\infty} = ||w_{2} T_{mp}||_{\infty} = ||w_{2} T_{ap}^{-1}||_{\infty} \geq ||w_{2}(p)|| \frac{z+p}{z-p}|$$

4. The Waterbed effect: Let  $M_1 := \max_{\omega_1 \le \omega \le \omega_2} |S(j\omega)|$  and  $M_2$  the maximum magnitude over all frequencies, i.e.  $||S||_{\infty}$ . If L has a zero z in the RHP then there exist  $c_1$  and  $c_2$  depending on  $\omega_1, \omega_2$  and z such that

131 1			
131 M2 M,		~	1151100
M, +			
	47	wz	

cilog Mi + a log Ma = log / Say (251/ 20

5= 1+PC

t - IC

Maximon modulos principle

uses Poisson integral firmula; proof in section 6.2 of DFJ

5. The area formula: Let  $p_i$  denote the set of poles of L in RHP.

$$\int_{0}^{\infty} \log |S(j\omega)| \, d\omega = \pi \log e \left( \sum Re \left\{ p_i \right\} \right)$$

**Exercise 5.1.** Let  $\omega$  be a frequency such that  $j\omega$  is not a pole of P. Suppose that  $\epsilon := |S(j\omega)| < 1$ at this frequency. Derive a lower bound for  $|C(j\omega)|$  that blows up as  $\epsilon \to 0$ .

$$\begin{aligned} \epsilon &= \frac{1}{11 + Pc1} \implies \frac{1}{\epsilon} = 11 + Pc1 \leq 1 + \frac{1}{P(1c)} \\ \implies 1c1 \geq \frac{1}{1P1} \frac{(1-\epsilon)}{\epsilon} \end{aligned}$$

#### **DFT** Chapter 7: Loopshaping 6

The idea is to construct a **stable** loop transfer function L such that

$$|||W_1S| + |W_2T|||_{\infty} < 1$$

is satisfied and then get C = L/P. To do this, require ... (typics)

for low brequency	1w, 1 >> 1 > 1w21,	$ L  > \frac{ w_i }{ w_i }$	
for high begrency	$ \omega_{1}  < 1 < <  \omega_{2} $	$ L  > \frac{ W_i }{ - W_2 }$ $ L  < \frac{ - W_i }{ W_2 }$	
		1 ~2	

and shape L accordingly, until you get an L that satisfies robust stability (make sure it's feedback is stable). In general, if L rolls off with more than 40 dB/dec at the crossover frequency the closed loop system is likely to be unstable.

#### Some Brief Exercises 7

Translate the following specifications for the closed-loop system.

1. The effect of the plant input disturbances at the plant output is reduced to  $\varepsilon$  times their open y = P(d - Cy)  $\Rightarrow Hyd = \frac{P}{1 + PC}$ loop magnitude for  $0 \le \omega \le 1$  rad/s. ત

17/ <0.01 w≥10 mad

2. Sensor noise should be reduced at the plant output by at least 40dB for  $\omega \ge 10 \text{ rad/s}$ y = -PC(n+y)

$$y = -PC(n+y) \qquad |T| = 0.01 \quad w \ge 10 \text{ rad} \qquad |S+T| = 1 \\ |S+T| = 1 \\ |S+T| = |S| \\ |S| + |T| \ge |S| \\ \Rightarrow \qquad |S| \ge 1 - 1 \\ \text{Suppose that } |S(jw)| \le 1.46 \text{ for all } \omega. \text{ Show that this is sufficient to ensure a phase margin of} \\ \text{at least } 40^{\circ}. \qquad B_{1} = 1 \\ \Rightarrow \qquad S_{2} = 1 \\ \Rightarrow \qquad$$

$$\frac{1}{|S(j_w)|} = |I+L| \qquad Re$$

$$\frac{1}{|S(j_w)|} = \inf_{w} |I+L(j_w)| \ge \frac{1}{1.46}$$

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d=R sin 9 => q = Arcsin (1/146) = ayr = 43.23

14

|S+T| = 1

151+1+1≥ |S++| = 1

# 8 Discrete Time Systems

### z-Transform and Difference Equations

- 1. Definition  $Z{f(k)} = \sum_{k=0}^{\infty} f(k)z^{-k}$
- 2. Get z transfer function from difference equations

$$Z\{f(k-1)\} = z^{-1}F(z), \text{ if } Z\{f(k)\} = F(z)$$

For difference equations

$$y(k) = -a_1 y(k-1) - a_2 y(k-2) + b_0 u(k) + b_1 u(k-1),$$

the z transfer function is

$$H(z) = \frac{Y(z)}{U(z)} = \frac{b_0 + b_1 z^{-1}}{1 + a_1 z^{-1} + a_2 z^{-2}}.$$

For state space system

$$\begin{cases} x(k+1) &= A_d x(k) + B_d u(k) \\ y(k) &= C_d x(k) + D_d u(k) \end{cases},$$

the z transfer function is  $H(z) = C_d(z \cdot I - A_d)^{-1}B_d + D_d$ .

3. Final value theorem of z transfer function:

$$\lim_{k \to \infty} h(k) = \lim_{z \to 1} (1 - z^{-1}) H(z).$$

4. The stable area of discrete systems is |z| < 1 in the z-plane. You can use the same skills as the continuous systems to design compensator, relocate poles, set up observers, etc.

### **Continuous to Discrete Transformation**

- 1. s-transfer function to z-transfer function by substitute every s as  $\frac{2}{T} \frac{1-z^{-1}}{1+z^{-1}}$ . This can be done by MATLAB command as c2d(tf, T, 'tustin') which is based on bilinear approximation (Tustin's method).
- 2. State space case. There are three methods mentioned in class. Assume a continuous system is

$$\dot{x} = A_c x + B_c u$$
$$y = C_c x + D_c u$$

(a) Forward Difference

$$x(k+1) = (I + TA_c)x(k) + TB_cu(k)$$
$$y(k) = C_cx(k)$$

Note: c2d(sys, T, 'ZOH') is not totally equal to forward difference method.

(b) Backward Difference

$$x(k+1) = (I - TA_c)^{-1}x(k) + (I - TA_c)^{-1}TB_c u(k)$$
$$y(k) = C_c x(k)$$

(c) Bilinear Approximation (Tustin's Method)

$$x(k+1) = (I - 0.5TA_c)^{-1}(I + 0.5TA_c)x(k) + 0.5T(I - 0.5TA_c)^{-1}(I + 0.5TA_c)B_cu(k)$$
$$y(k) = C_c x(k)$$

# 9 Additional Notes