


CDS 101: Lecture 3.1 Stability and Performance

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Goals:


- Describe different types of stability for an equilibrium point
- Explain the difference between local/global stability, and related concepts
- Describe performance measures for (controlled) systems, including transients and steady state response

Reading:

- Åström and Murray, *Analysis and Design of Feedback Systems*, Ch 3


Review from Last Week

Model = state, inputs, outputs, dynamics



$$\frac{dx}{dt} = f(x, u)$$

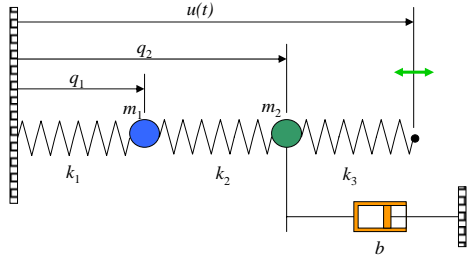
$$y = h(x)$$



$$x_{k+1} = f(x_k, u_k)$$

$$y_{k+1} = h(x_{k+1})$$

Principle: Choice of model depends on the questions you want to answer



```
function dydt = f(t,y, k1, k2,
k3, m1, m2, b, omega)
u = 0.00315*cos(omega*t);
dydt = [
  y(3);
  y(4);
  -(k1+k2)/m1*y(1) +
    k2/m1*y(2);
  k2/m2*y(1) - (k2+k3)/m2*y(2)
  - b/m2*y(4) + k3/m2*u ];
```

Today: Stability and Performance

Goal #1: Stability

- Check if *closed loop* response is stable

$\dot{x} = f(x, u) \quad u = k(x)$

↑ control law
↑ system input

Goal #2: Performance

- Look at ability to track changes in reference and reject disturbances

$\dot{x} = f(x, k(x, r), d)$
 $y = h(x)$

↑ disturbance
↑ reference
↑ measured output

Goal #3: Robustness (later)

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Phase Portraits (2D systems only)

Phase plane plots show 2D dynamics as *vector fields & stream functions*

- Plot $f(x)$ as a vector on the plane; stream lines follow the flow of the arrows

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ -x_1 - x_2 \end{bmatrix}$$

```
phaseplot('dosc', ...
[-1 1 10], [-1 1 10], ...
boxgrid([-1 1 10], [-1 1 10]));
```

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Equilibrium Points

Equilibrium points represent stationary conditions for the dynamics

The *equilibria* of the system $\dot{x} = f(x)$ are the points x_e such that $f(x_e) = 0$.

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ \sin(x_1) \end{bmatrix} \Rightarrow x_e = \begin{bmatrix} \pm n\pi \\ 0 \end{bmatrix}$$

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Stability of Equilibrium Points

An equilibrium point is:

Asymptotically stable if all nearby initial conditions converge to the equilibrium point

- Equilibrium point is an *attractor or sink*

Unstable if some initial conditions diverge from the equilibrium point

- Equilibrium point is a *source (or saddle)*

Stable if initial conditions that start near the equilibrium point, stay near

- Equilibrium point is a *center*

$$\lim_{t \rightarrow \infty} x(t) = x_e \quad \forall \|x(0) - x_e\| < \varepsilon$$

$$\lim_{t \rightarrow \infty} \|x(t)\| = \infty \text{ for some } x(0)$$

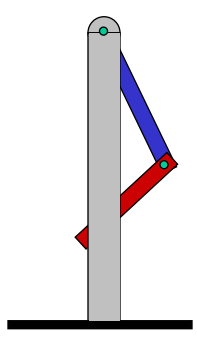
$$\|x(t) - x_e\| < \varepsilon \quad \forall t, \|x(0) - x_e\| < \delta_\varepsilon$$

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Example #1: Double Inverted Pendulum

Two series coupled pendula

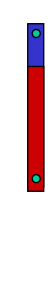
- States: pendulum angles (2), velocities (2)
- Dynamics: $F = ma$ (balance of forces)
- Dynamics are very nonlinear



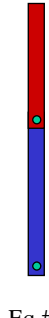
Eq #1




Eq #2



Eq #3



Eq #4



Stability of equilibria

- Eq #1 is stable
- Eq #3 is unstable
- Eq #2 and #4 are unstable, but with some stable "modes"

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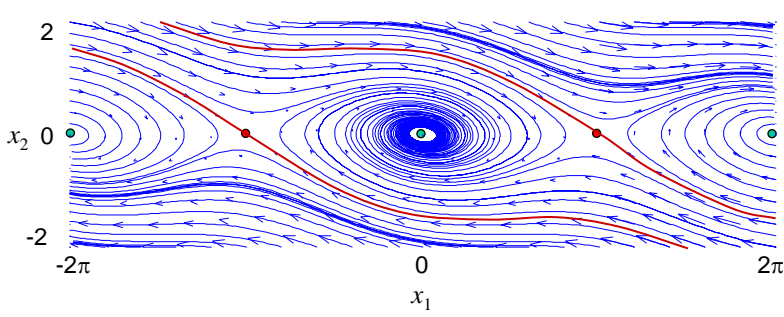
Local versus Global Behavior

Stability is a *local* concept

- Equilibrium points define the local behavior of the dynamical system
- Single dynamical system can have stable *and* unstable equilibrium points

Region of attraction

- Set of initial conditions that converge to a given equilibrium point



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Example #2: Predator Prey (ODE version)

Continuous time (ODE) version of predator prey dynamics:

$$\dot{R} = r_R R \left(1 - \frac{R}{K}\right) - \frac{aRF}{1 + aRT_h}$$

$$\dot{F} = r_F F \left(1 - \frac{F}{kR}\right)$$

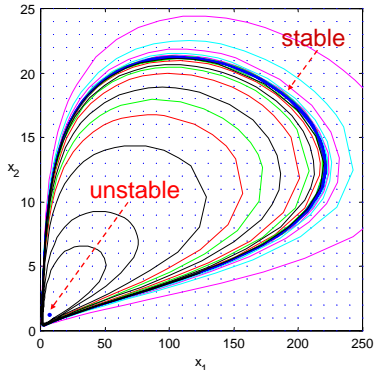
- Continuous time (ODE) model
- MATLAB: predprey.m (from web page)

Equilibrium points

- $\sim(6.5, 1.3)$: unstable \Rightarrow no steady state population

Invariant curves (3)

- Start on curve, stay on curve
- "Limit cycle" \Rightarrow population of each species oscillates over time
- This is a *global* feature of the dynamics (not local to an equilibrium point)




The phase plane plot shows trajectories in the (x_1, x_2) plane. The horizontal axis is x_1 (0 to 250) and the vertical axis is x_2 (0 to 25). A red dashed line indicates an unstable equilibrium point near $(6.5, 1.3)$. A blue dashed line indicates a stable equilibrium point. Trajectories are shown as colored curves that oscillate around the stable equilibrium point.

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Input/Output Performance

Return to system with inputs

- How does system response to changes in input values?



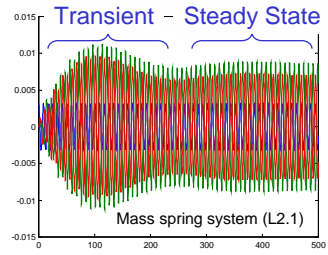
The block diagram shows a system with a step input (red square wave) and an oscillatory output (blue sine wave). The system is represented by a photograph of a white SUV.

Transient response:

- What happens right after a new input is applied

Steady state response:

- What happens a long time after the input is applied



The plot shows the response of a mass spring system (L2.1) to a step input. The vertical axis ranges from -0.015 to 0.015, and the horizontal axis ranges from 0 to 500. The plot is divided into a "Transient" region (0 to 300) and a "Steady State" region (300 to 500). The transient response shows a decaying oscillation, and the steady state response shows a sustained oscillation.

Stability vs input/output performance


- Systems that are close to instability typically exhibit poor input/output performance
- Nearly unstable systems (slow convergence) often exhibit "ringing" (highly oscillatory response to [non-periodic] inputs)

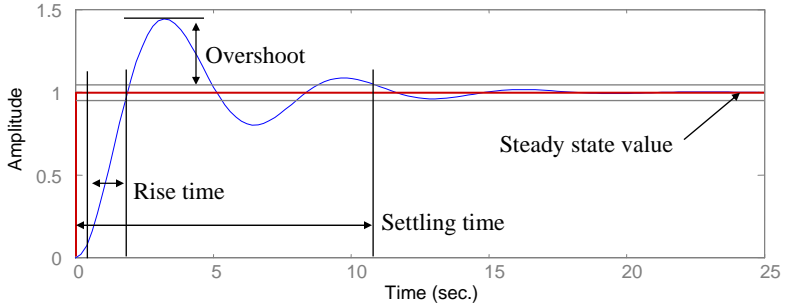
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Step Response

Output characteristics in response to a “step” input

- Rise time: time required to move from 5% to 95% of final value
- Overshoot: ratio between amplitude of first peak and steady state value
- Settling time: time required to remain w/in $p\%$ (usually 2%) of final value
- Steady state value: final value at $t = \infty$



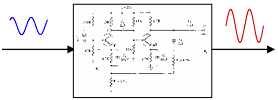


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Frequency Response

Measure the *steady state* response of the system to sinusoidal input

- Example: audio amplifier – would like consistent (“flat”) amplification between 20 Hz & 20,000 Hz
- Individual sinusoids are good *test signals* for measuring performance in many systems (eg, seasonal cycles in temperature)

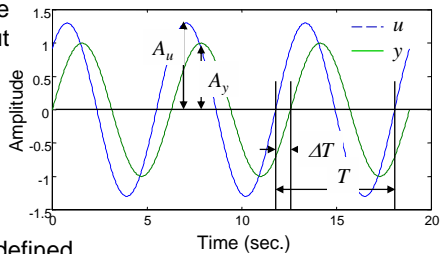


Approach: plot input and output, measure *relative* amplitude and phase

- Use MATLAB or SIMULINK to generate response of system to sinusoidal output
- Gain = A_y/A_u
- Phase = $2\pi \cdot \Delta T/T$

May not work for *nonlinear* systems

- System nonlinearities can cause *harmonics* to appear in the output
- Amplitude and phase may not be well-defined
- For *linear* systems, frequency response is always well defined (week 6)



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Second Order Systems

Important class of systems in many applications areas

$$\ddot{q} + 2\zeta\omega_0\dot{q} + \omega_0^2q = u \quad \longleftrightarrow \quad \frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\omega_0^2 & 2\zeta\omega_0 \end{bmatrix} x + \begin{bmatrix} 0 \\ \frac{1}{m} \end{bmatrix} u$$

- Analytical formulas exist for overshoot, rise time, settling time, etc
- Frequency response can also be analytically derived

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Summary: Stability and Performance

Key topics for this lecture

- Stability of equilibrium points
- Local versus global behavior
- Performance specification via step and frequency response

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