



## CDS 101: Lecture 3.1 Stability and Performance



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13 October 2003

### Goals:

- Describe different types of stability for an equilibrium point
- Explain the difference between local/global stability, and related concepts
- Describe performance measures for (controlled) systems, including transients and steady state response

### Reading:

- Åström and Murray, *Analysis and Design of Feedback Systems*, Ch 3

## Review from Last Week

Model = state, inputs, outputs, dynamics



$$\frac{dx}{dt} = f(x, u)$$

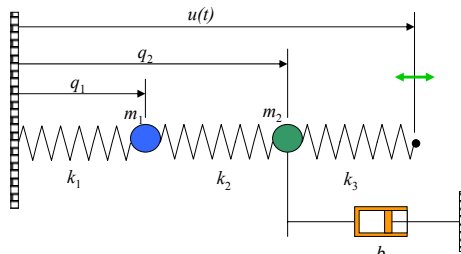
$$y = h(x)$$



$$x_{k+1} = f(x_k, u_k)$$

$$y_{k+1} = h(x_{k+1})$$


**Principle:** Choice of model depends on the questions you want to answer



```
function dydt = f(t,y, k1, k2,
k3, m1, m2, b, omega)
u = 0.00315*cos(omega*t);
dydt = [
    y(3);
    y(4);
    -(k1+k2)/m1*y(1) +
        k2/m1*y(2);
    k2/m2*y(1) - (k2+k3)/m2*y(2)
    - b/m2*y(4) + k3/m2*u ];
```

### Today: Stability and Performance

**Actuate**  
Gas Pedal



**Sense**  
Vehicle Speed

**Compute**  
Control "Law"

**Goal #1: Stability**

- Check if *closed loop* response is stable

**Goal #2: Performance**

- Look at ability to track changes in reference and reject disturbances

**Goal #3: Robustness (later)**

$\dot{x} = f(x, u) \quad u = k(x)$   
↑ ↑  
control law  
system input

$\dot{x} = f(x, k(x, r), d)$   
 $y = h(x)$   
↑ ↑  
disturbance  
reference  
measured output

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### Systems of Ordinary Differential Equations (ODEs)

**Systems of coupled nonlinear eqns (closed loop ⇒ no inputs for now)**

Component form

 $\dot{x}_1 = f_1(x_1, x_2, \dots, x_n)$   
 $\dot{x}_2 = f_2(x_1, x_2, \dots, x_n)$   
 ...  
 $\dot{x}_n = f_n(x_1, x_2, \dots, x_n)$

Vector form

 $\dot{x} = f(x)$   
 $x = \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix} \in \mathbb{R}^n$

MATLAB form

```

function dxdt = f(t,x)
dxdt= [
    f1(x);
    f2(x);
    ...
    fn(x);
];
                    
```

**Example: damped oscillator**

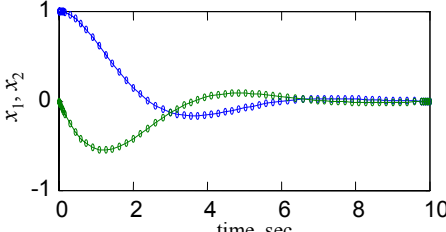
 $\dot{x}_1 = x_2$   
 $\dot{x}_2 = -x_1 - x_2$   
 $f(x) = \begin{bmatrix} x_2 \\ -x_1 - x_2 \end{bmatrix} \in \mathbb{R}^2$

```
ode45('dosc', [0 10], [1 0]);
```

↑  
function

↑  
time interval

↑  
initial condition



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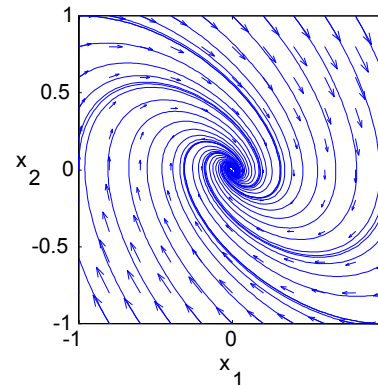
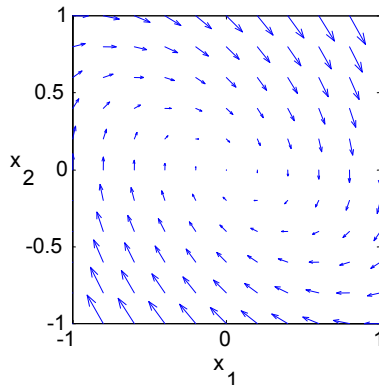
## Phase Portraits (2D systems only)

Phase plane plots show 2D dynamics as *vector fields* & *stream functions*

- Plot  $f(x)$  as a vector on the plane; stream lines follow the flow of the arrows

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ -x_1 - x_2 \end{bmatrix}$$

```
phaseplot('dosc', ...
  [-1 1 10], [-1 1 10], ...
  boxgrid([-1 1 10], [-1 1 10]));
```



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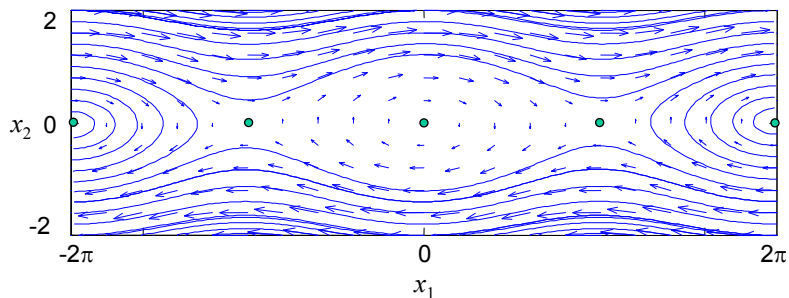
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## Equilibrium Points

Equilibrium points represent stationary conditions for the dynamics

The *equilibria* of the system  $\dot{x} = f(x)$  are the points  $x_e$  such that  $f(x_e) = 0$ .

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ \sin(x_1) \end{bmatrix} \Rightarrow x_e = \begin{bmatrix} 0 \\ \pm n\pi \end{bmatrix}$$



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### Stability of Equilibrium Points

**An equilibrium point is:**

**Asymptotically stable** if all nearby initial conditions converge to the equilibrium point

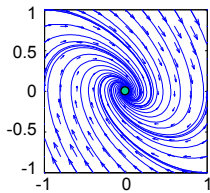
- Equilibrium point is an *attractor or sink*

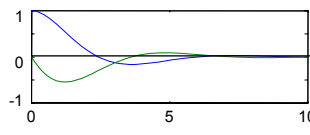
**Unstable** if some initial conditions diverge from the equilibrium point

- Equilibrium point is a *source (or saddle)*

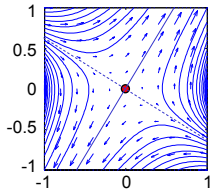
**Stable** if initial conditions that start near the equilibrium point, stay near

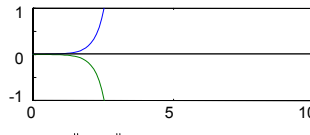
- Equilibrium point is a *center*



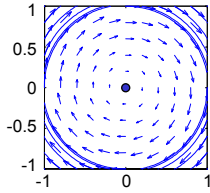


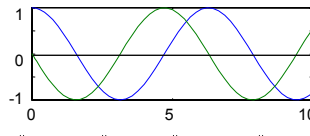
$$\lim_{t \rightarrow \infty} x(t) = x_c \quad \forall \|x(0) - x_c\| < \varepsilon$$





$$\lim_{t \rightarrow \infty} \|x(t)\| = \infty \text{ for some } x(0)$$





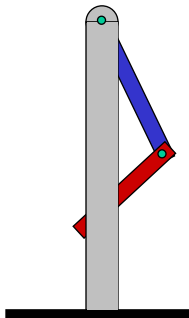
$$\|x(t) - x_c\| < \varepsilon \quad \forall t, \|x(0) - x_c\| < \delta_c$$

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### Example #1: Double Inverted Pendulum

**Two series coupled pendula**

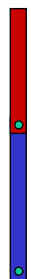

- States: pendulum angles (2), velocities (2)
- Dynamics:  $F = ma$  (balance of forces)
- Dynamics are very nonlinear



Eq #1      Eq #2

Eq #3      Eq #4

**Stability of equilibria**

- Eq #1 is stable
- Eq #3 is unstable
- Eq #2 and #4 are unstable, but with some stable "modes"

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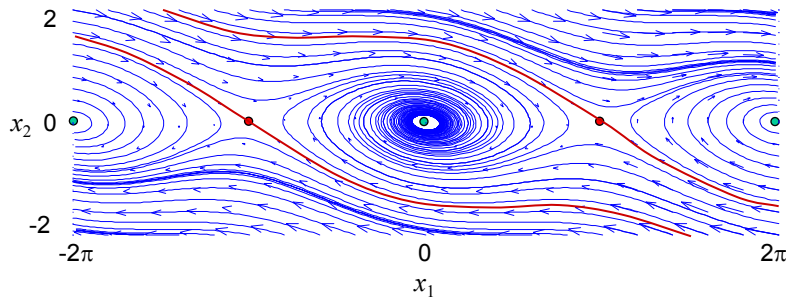
## Local versus Global Behavior

### Stability is a *local* concept

- Equilibrium points define the local behavior of the dynamical system
- Single dynamical system can have stable *and* unstable equilibrium points

### Region of attraction

- Set of initial conditions that converge to a given equilibrium point



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## Example #2: Predator Prey (ODE version)

### Continuous time (ODE) version of predator prey dynamics:

$$\dot{x}_1 = b_r x_1 - a x_1 x_2$$

$$\dot{x}_2 = a x_1 x_2 - d_f x_2$$

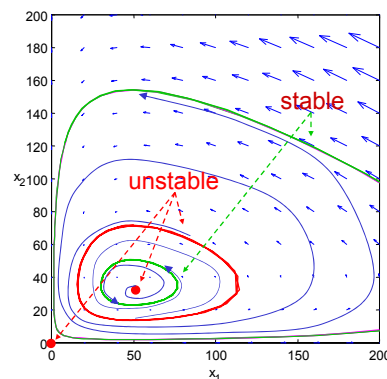
- Continuous time (ODE) model
- MATLAB: predprey.m (from web page)

### Equilibrium points (2)

- $(0,0)$ : unstable  $\Rightarrow$  species don't die out
- $\sim(50,35)$ : unstable  $\Rightarrow$  no steady state population

### Invariant curves (3)

- Start on curve, stay on curve
- "Limit cycle"  $\Rightarrow$  population of each species oscillates over time
- This is a *global* feature of the dynamics (not local to an equilibrium point)



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## Input/Output Performance

### Return to system with inputs

- How does system response to changes in input values?



### Transient response:

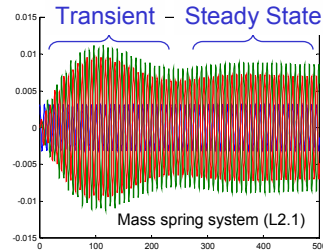
- What happens right after a new input is applied

### Steady state response:

- What happens a long time after the input is applied

### Stability vs input/output performance

- Systems that are close to instability typically exhibit poor input/output performance
- Nearly unstable systems (slow convergence) often exhibit “ringing” (highly oscillatory response to [non-periodic] inputs)



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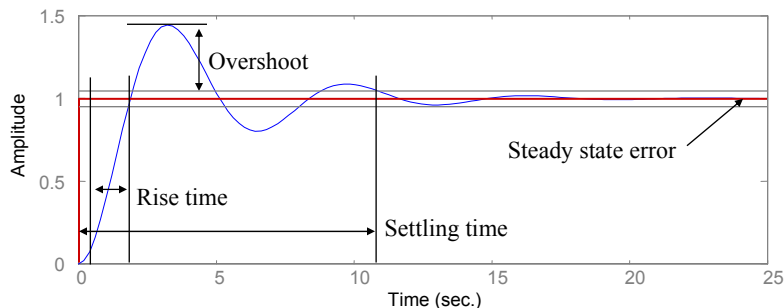
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## Step Response

### Output characteristics in response to a “step” input

- Rise time: time required to move from 10% to 90% of final value
- Overshoot: ratio between amplitude of first peak and steady state value
- Settling time: time required to remain w/in  $p\%$  (usually 2%) of final value
- Steady state error: residual error at  $t = \infty$



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### Frequency Response

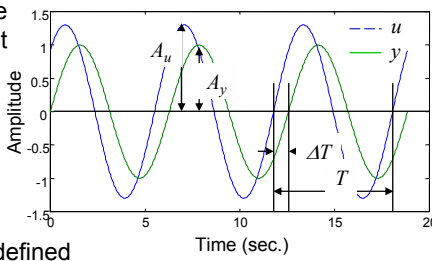
**Measure the steady state response of the system to sinusoidal input**

- Example: audio amplifier – would like consistent (“flat”) amplification between 20 Hz & 20,000 Hz
- Individual sinusoids are good *test signals* for measuring performance in many systems (eg, seasonal cycles in temperature)



**Approach: plot input and output, measure relative amplitude and phase**

- Use MATLAB or SIMULINK to generate response of system to sinusoidal output
- Gain =  $A_y/A_u$
- Phase =  $2\pi \cdot \Delta T/T$

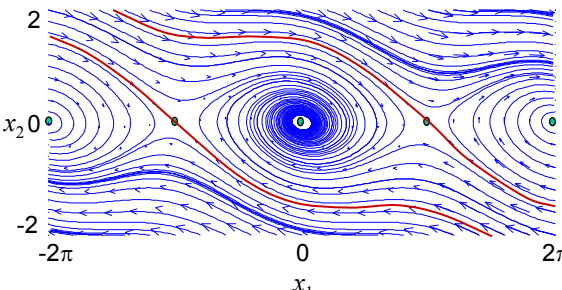


**May not work for nonlinear systems**

- System nonlinearities can cause *harmonics* to appear in the output
- Amplitude and phase may not be well-defined
- For *linear* systems, frequency response is always well defined (week 6)

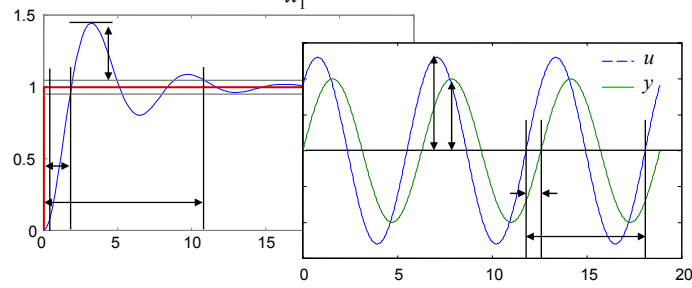
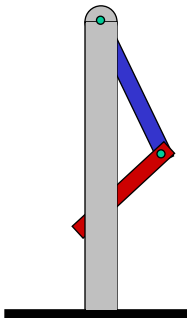
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### Summary: Stability and Performance



**Key topics for this lecture**

- Stability of equilibrium points
- Local versus global behavior
- Performance specification via step and frequency response

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```

% I3_1_stability.m - MATLAB source code for Lecture 3.1
% RMM, 11 Oct 03
% Required files: oscillator.m, pendulum.m, saddle.m, predprey.m
% Systems of ODEs: damped oscillator example (simulation + phase portrait)
% This example uses the ODEs in dampedosc.m, available separately.
% Simulation of the damped oscillator
figure(1); ode45('oscillator', [0 10], [1 0]);
xlabel('x_1'); ylabel('x_2', 'Rotation', 0);
% Generate a vector plot for the damped oscillator
figure(2); phaseplot('oscillator', [-1 1 10], [-1 1 10]);
% Generate a phase plot for the damped oscillator
figure(3); phaseplot('oscillator', ...
[-1 1 10], [-1 1 10], boxgrid([-1 1 10], [-1 1 10]));
% Equilibrium points
% This example uses the undamped pendulum to illustrate multiple equilibrium
% points.
m = 1; l = 1; b = 0; g = 1; % parameters for undamped pendulum
figure(1); phaseplot('pendulum', ...
[-2*pi 2*pi 20], [-2 2 10], ...
boxgrid([-2*pi 2*pi 20], [-2 2 10]), 10, ...
m, l, b, g);
% Stability definitions
% This set of plots illustrates the various types of equilibrium points.
% Asy stable
m = 1; b = 1; k = 1; % default values
figure(1); phaseplot('oscillator', [-1 1 10], [-1 1 10], ...
boxgrid([-1 1 10], [-1 1 10]), 10, m, b, k);
print -dmeta asystable.wmf
[x,t] = ode45('oscillator', [0 10], [1 0], [], m, b, k);
figure(2); plot(x, t);
% Unstable
% Note: parameter values need tweaking to generate plot from Lecture
figure(1); phaseplot('saddle', [-1 1 10], [-1 1 10], ...
boxgrid([-1 1 10], [-1 1 10]), 1);
[x,t] = ode45('saddle', [0 10], [1 0]);
figure(2); plot(x, t); axis([0 10 -100 100]);
% Stable ist
% zero damping
m = 1; b = 0; k = 1;
figure(1); phaseplot('oscillator', [-1 1 10], [-1 1 10], ...
boxgrid([-1 1 10], [-1 1 10]), 10, m, b, k);

```

```

[x,t] = ode45('oscillator', [0 10], [1 0], [], m, b, k);
figure(2); plot(x, t);
% Local versus global behavior
% This example uses a damped pendulum to illustrate regions of attraction
% The coloring in the powerpoint slide is done by hand (ugh)
m = 1; l = 1; b = 0.5; g = 1; % parameters for damped pendulum
figure(1); phaseplot('pendulum', ...
[-2*pi 2*pi 20], [-2 2 10], ...
boxgrid([-2*pi 2*pi 20], [-2 2 20]), 10, ...
m, l, b, g);
% Predator prey example
% The code implementing this is in the file predprey.m.
global pp K pp_xeq;
pp_K = [0, 0]; pp_xeq = [50, 35]; % declare control variables
% turn control off
% Generate a simulation of the system
figure(1);
ode45('predprey', [0 100], [10 10]);
figure(2); phaseplot('predprey', ...
[0 200 10], [0 200 10], ...
[50, 2; 50, 35; 50, 50.8; 50, 71.706], ... % numerically determined
50);
% oscillator.m - ODEs for a 2D oscillator
% RMM, 11 Oct 03
% This function gives the dynamics for a damped oscillator. The states
% of the system are
% x(1) position
% x(2) velocity
% The default parameters for the system are given by
% m = 1 mass, kg
% k = 1 spring constant, N/m
% b = 1 damping constant, N-sec/m
% This corresponds to a fairly heavily damped oscillator.
function dx=oscillator(t, x, flags, m, b, k)
if (nargin < 4) m = 1; end;
if (nargin < 5) b = 1; end;
if (nargin < 6) k = 1; end;
dx=[x(2); -k/m*x(1) - b/m*x(2)];

```